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could

$$J\dot{\omega} + k(\omega - \bar{\omega}) = -\tilde{T} \sin 2\pi t/t_0$$

Setting  $x = 2\pi t/t_0$  and defining the constant  $m$  as  $m = 2\pi J/k t_0$ , the solution to this DE is:  
 $(\omega - \bar{\omega}) k/\tilde{T} = C \exp(-x/m) - \frac{1}{\sqrt{1+m^2}} \sin(x - \tan^{-1} m)$ .

Considering only the steady-state component of the RHS, the amplitude of the speed excursion about  $\bar{\omega}$  is thus

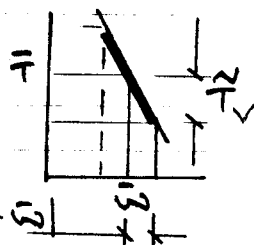
$$|\omega - \bar{\omega}|_{\max} = \frac{\tilde{T}}{k} \cdot \frac{1}{\sqrt{1+m^2}}$$

and the corresponding torque amplitude on the meter line is

$$\tilde{T}_V = k |\omega - \bar{\omega}|_{\max} = \tilde{T} \frac{1}{\sqrt{1+m^2}}$$

$$\text{Hence } \frac{\tilde{T}_V}{\tilde{T}} = [1 + (2\pi J/k t_0)^2]^{-1/2}$$

QED.



6 First ascertain the effective torque for a sinusoid of amplitude  $\tilde{T}$  and mean  $\bar{T}$ :

$$T_e \cdot t_0 = \int_0^{t_0} (\bar{T} + \tilde{T} \sin \frac{2\pi t}{t_0})^n dt$$

with  $n=2$

$$T_e^2 t_0 = \int_0^{t_0} (\bar{T}^2 + 2\bar{T}\tilde{T} \sin \frac{2\pi t}{t_0} + \frac{1}{2}\tilde{T}^2 (1 - \cos \frac{4\pi t}{t_0})) dt$$

$$= (\bar{T}^2 + \frac{1}{2}\tilde{T}^2) t_0$$

ie.  $T_e = \sqrt{(\bar{T}^2 + \frac{1}{2}\tilde{T}^2)}$  or use (A2)

Of the 100 M motor, with  $T_f = 120$  Nm were to drive the punch, with no inertia,  $\bar{T} = 110$ ,  $\tilde{T} = 100$  Nm, &

$$T_e = \sqrt{(110^2 + \frac{1}{2} \times 100^2)} = 131 \text{ Nm.}$$

- clearly not feasible.

With a view to solving the rackets of the previous problem, estimate the slope  $k$  at the operating point.

The torque-speed equation is:

$$(a) \quad T = T_b / (1 + (s-s_b)^2 (a/s - b s^2)) ; \quad s = 1 - \frac{n}{N_s} = 1 - \frac{\omega}{\omega_s}$$

Differentiating this to get the slope:

$$dT/d\omega = (ds/d\omega) (dT/ds) ; \quad ds/d\omega = -1/\omega_s$$

$$= -(T^2/\omega_s T_b) (s_b - s) [ a(s_b + s)/s^2 + 2bs(s_b - 2s) ]$$

- the second term in the [ ] is negligible, so

$$k = -dT/d\omega = \frac{T^2}{\omega_s T_b} \left( \left( \frac{s_b}{s} \right)^2 - 1 \right)$$

At the mean operating point,  $\bar{T}$  of 110 Nm, the corresponding speed is found to be 1473 rpm, from (a) which corresponds to a slip of 0.018. So at this point:

$$k = 1.500 \times 110^2 \left[ \left( \frac{0.179}{0.018} \right)^2 - 1 \right] / 50\pi \times 348$$

$$= 32.5 \text{ Nms (rad)}$$