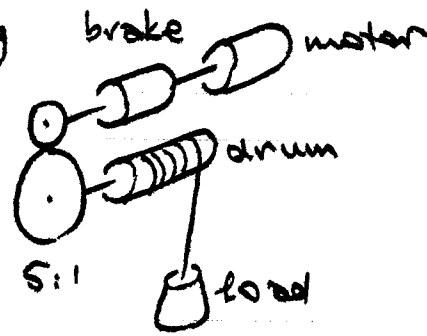


- 1 Assume stopping in range 59-60m
 \therefore Total load = $10^3 + 59.5 \times 0.5 = 1030$ kg
 Take steel density 7.8 t/m³.
 Take PE datum at 60m level.
 $I_{\text{drum}} = 120 \times 0.11^2 = 1.45$ kgm²
 $m_{\text{wheel}} = \pi \times 0.36 \times 0.05 \times 0.016 \times 7.8 \times 10^3 = 7.06$ kg
 $\therefore I_{\text{wheel}} = 7.06 \left(\frac{0.36}{2}\right)^2 = 0.23$ kgm²



- When the rope speed is 3m/s, $\omega_{\text{drum}} = 3 / \left(\frac{0.25}{2}\right) = 24$ rad/s
 $\therefore \omega_{\text{motor}} = 5 \omega_{\text{drum}} = 120$ rad/s
 The initial energy of the system is:
 PE load = $W \cdot h = 1030 \times 9.81 \times 1 = 10.10$ kNm
 KE load = $\frac{1}{2} m v^2 = \frac{1}{2} \times 1030 \times 3^2 = 4.64$ "
 KE drum shaft = $\frac{1}{2} I \omega^2 = \frac{1}{2} (1.45 + 0.23) \times 24^2 = 0.48$ "
 KE motor shaft = $\frac{1}{2} I \omega^2 = \frac{1}{2} \times 0.3 \times 120^2 = 2.16$ "
 \therefore work done by brake ($E_{\text{final}} = 0$), $W = 17.38$ "

Rotation of brake (i.e. motor shaft) = $5 \times \frac{\Delta \theta}{5} = 5 \times 0.125 = 40$ rad
 $\bar{\omega} = \omega_0 / 2 = 60$ rad/s = $\Delta \theta / \Delta t$ $\therefore \Delta t = 40 / 60 = 2/3$ sec
 \therefore Brake torque = $W / \Delta \theta = 17.38 \times 10^3 / 40 = 435$ Nm
 $\&$ mean power = $W / \Delta t = 17.38 \times 10^3 / (2/3) = 26.1$ kW

2. $T_0 = \sum T = \sum \eta M = \sum \eta (\lambda P_0)$ from (12) = $P_0 \sum \lambda \eta$
 $\therefore \eta_0 = T_0 / P_0 = \sum \lambda \eta$ Q.E.D.
 From (vii)
 $S_0 = \frac{M}{T_0} \frac{\partial T_0}{\partial u} / P_0$ where, from above, $T_0 = P_0 \sum \lambda \eta$.
 $= \frac{M}{T_0} P_0 \sum \lambda \frac{d\eta}{du} = \frac{M P_0}{T_0} \sum \lambda \frac{m \eta^2}{u}$ (since $\eta = \frac{k}{u - \delta \Delta u}$)
 - provided geometrically similar shes.
 $= m \sum \lambda \eta^2 / \sum \lambda \eta = \sum \lambda \eta S / \eta_0$ using (8) Q.E.D.

3, 4, 5, 6 The solutions to these follow along the lines of the worked example in the Notes. The output of program 'Brakes' is shown overleaf. Special points to note are as follows.

3. Straight forward - actuation force and width are given - no scaling should be necessary. Rotation axis is immaterial, since asymmetric.

4. This is a most unusual brake. The top half is a mirror image of the lower two shes, so results can be gotten

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