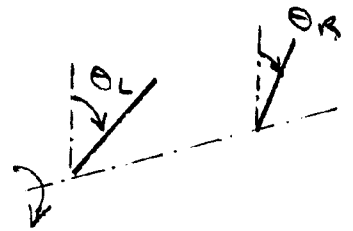


8
cont'd

TORSION
Compatibility

If θ_L and θ_R are the angular deflexions of the tube in way of L and R discs, the rotating of the tube (relative) between the discs, θ_t , is

$$\theta_t = \theta_L - \theta_R$$

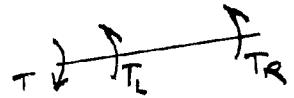


Equilibrium

The stress momentants in the bands are T_L & T_R , where

$$T_L + T_R = T = 10 \text{ Nm}$$

Note that the relative rotation of the discs between the discs is due to the torque transmitted along the tube between them, that is T_R , as may be seen by a R.H free body.



Constitutive Laws

Discs : $T_L = k_d \theta_L$; $T_R = k_d \theta_R$
where k_d is the disc stiffness

$$k_d = \frac{4\pi b G}{1/r_1^2 - 1/r_2^2} = \frac{4\pi \times 20 \times 2}{2(1/0.5^2 - 1/0.5^2)} \times 10^{-3} = 942 \text{ Nm/rad.}$$

Tube : from torsion eqn.

$$k_t = \frac{T}{\theta} = \frac{3T}{L} = \frac{2E3}{2(1+0.35)} \times \frac{\pi}{32} (60^4 - 54^4) = 540 \text{ Nm/rad.}$$

$$T_R = k_t \theta_t$$

Solution

$$T_L = 7.3 \text{ Nm} ; T_R = 2.7 \text{ Nm}$$

In terms of the shear stress in the band, the torsional stress momentant is

$$T (=T_L \text{ or } T_R) = \tau A r = \tau \times 2\pi r^2 b ; r = 30 \text{ mm.}$$

So the tangential shear components are

$$T_L = 65 \text{ kPa} \quad T_R = 24 \text{ kPa.}$$

Evidently the left band is much the more heavily loaded. For this, from above

$$\tau_A = 65 \text{ kPa} \quad \tau_c = 65 \text{ kPa}$$

$$\therefore \tau = \sqrt{65^2 + 65^2} = 95 \text{ kPa}$$

and the safety factor for the bands is

$$n = \frac{\hat{\tau}}{\tau} = 100/95 = 1.05.$$