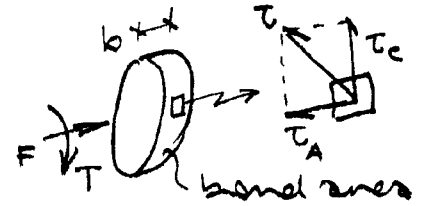


8

Approach: Find the actual shear stress, τ , in each band (left L and right R as sketched).

τ is made up of two components - an axial component τ_A due to F , and a circumferential component τ_C due to T - which may be combined vectorially.

Assuming τ_A and τ_C are both uniform, their stress components may be found from F and T individually - though in each case the problem is indeterminate.

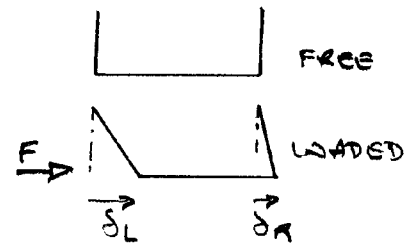


AXIAL

Compatibility

If δ_L and δ_R are the deflections of the discs in an axial direction, the contraction of the tube, δ_t is

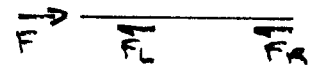
$$\delta_t = \delta_L - \delta_R.$$



Equilibrium

The stress resultants in the bands are F_L and F_R , and

$$F_L + F_R = F = 500 \text{ N}.$$



Note that the tube experiences the compressive force F_R between the discs (consider RH free body)

Constitutive Laws note $G = E/(2(1+\nu))$

Discs: $F_L = k_d \delta_L$ and $F_R = k_d \delta_R$

where k_d is the disc stiffness

$$k_d = \frac{2\pi b G}{\ln(r_o/r_i)} = \frac{2\pi \times 20 \times 2}{2(1+0.5) \ln(60/50)} = 104 \frac{\text{N}}{\text{mm}}$$

Tube: $F_L = F_R = k_t \delta_t$

where k_t is the axial stiffness of tube

$$k_t = AE/L = \frac{\pi}{4} (60^2 - 50^2) \times 2 \times 10^3 / 600 = 17.91 \frac{\text{N}}{\text{mm}}$$

Solution

$$F_L = 261 \text{ N} \quad F_R = 239 \text{ N}$$

i.e. load just about equally shared since tube is 20 much stiffer than discs.

Dividing these by the band area

$$A = \pi D b = \pi \times 60 \times 20 \text{ mm}^2$$

gives the axial shear components:

$$\tau_L = 6.9 \text{ kPa} \quad \tau_R = 6.3 \text{ kPa}$$