

12

$d^* = (1/d_1 + 1/d_2)^{-1} = (1/10 - 1/10.1)^{-1} = 1010 \text{ mm}$ ← since internal.
 $E^* = (1.29^2/207 + 1.3^2/110)^{-1} = 78.8 \text{ GPa}$

For spheres:

$\sigma^3 = \frac{3Fd^*}{\pi E^*} \quad \hat{p} = \frac{3F}{2\pi d^*}$

∴ the interdependence of \hat{p} & F is non-linear
 So eliminate ' σ ' to obtain direct relation

$\sigma^3 = 9F^2 d^{*2} / 64 E^{*2} = 27 F^3 / 8 \pi^3 \hat{p}^3$

whence: $F = \frac{1}{24} (\pi \hat{p})^3 (d^*/E^*)^2 \quad (i)$

(a) Using equation (i) above, with $\hat{p} = 200 \text{ MPa}$

$F = \frac{1}{24} (\pi \times 200)^3 (1010 / 78.8 \times 10^3)^2 = 1.70 \text{ kN}$
 $\text{N}^3/\text{mm}^6 \quad \text{mm}^2 \quad \text{mm}^4/\text{N}^2$

(b) Both materials are ductile, so max. shear criterion is valid, i.e. $\hat{\sigma}_E = 0.6 \hat{p}$

brass $\hat{\sigma}_E = 0.6 \hat{p} \leq 240 \quad \therefore \hat{p} \leq 400 \text{ MPa}$

steel $\hat{\sigma}_E = 0.6 \hat{p} \leq 400 \quad \therefore \hat{p} \leq 667 \text{ MPa}$

Obviously brass is weaker & $\hat{p} = 400 \text{ MPa}$

Since $F \propto \hat{p}^3$, and \hat{p} is double that of (a), then

$F = 2^3 \times 1.70 = 13.6 \text{ kN}$

(c) $E^* = [1.29^2/207 + 1.21^2/120]^{-1} = 71.5 \text{ GPa}$

From above, if steel is weaker material at the contact, $\hat{p} = 667 \text{ MPa}$

considering the cast iron, if it fails, it will do so by direct compressive stress on the surface exceeding the compressive design stress. If CI is weaker then $\hat{p} \leq 350 \text{ MPa}$

So CI is in fact the weaker, so $\hat{p} = 350 \text{ MPa}$

& $F = \frac{1}{24} (\pi \times 350)^3 (1010 / 71.5 \times 10^3)^2 = 11.0 \text{ kN}$

13

$d_1 \approx d = d(1 - \frac{1}{2}c)$

$d_2 \approx d = d(1 + \frac{1}{2}c)$

i.e. clearance = $d_2 - d_1 = cd$

noting that, expressed this way, 'c' is dimensionless

so $d^* = (1/d_1 + 1/d_2)^{-1}$, $d_2 \ll d$ since inward
 $= (1/d(1-\frac{1}{2}c) - 1/d(1+\frac{1}{2}c))^{-1} = d/c$

- Note the implication of improving the fit, eliminate unwanted ' σ ' from cylinder equations

$\sigma^2 = 2F d^* / \pi L E^* \quad \hat{p} = 2F / \pi d L$

gives, $\hat{p} = \sqrt{[\frac{2}{\pi} \cdot \frac{F}{d^*} \cdot E^*]}$

Note the implication of this, \hat{p} is approx. the geometric mean of E^* and average pressure over the contact F/Ld^*

finally substituting for $d^* \Rightarrow \hat{p} = \sqrt{(2cFE^*/\pi dL)}$

